



CHARGE and CHASE

The UK FORESIGHT VEHICLE Research

on Extending HCCL Engine Operating Boundaries

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Contribution to the current research

Jaguar Cars - Trevor Wilson, Huiyu Fu, Simon Cryan, Andy Williams, Simon Rudolph, Shadi Gharahbaghi, Stan Wallace and Steve Richardson

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Johnson Matthey – David James and Sylvain Peucheret

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Vehicle LINK Projects CHARGE and CHASE

Presentation Outline

- Objectives
- Project outline
- Current status
- Future prospective

What is FORESIGHT VEHICLE

FORESIGHT VEHICLE is the UK's prime knowledge transfer network for the automotive industry involving collaboration between industry, academia and Government.

The R&D programme aims to promote technology and stimulate suppliers to develop market driven enabling technologies for motor vehicles.

The research within the programme is to ensure that future products and technologies meet social, economic and environmental goals, satisfying requirements for mobility, safety, performance, cost and desirability.



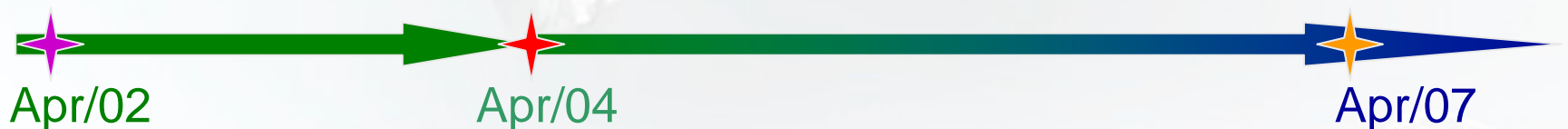
UK FORESIGHT projects on HCCI

CHARGE (Controlled Homogeneous Auto-ignition Reformed Gas Engine),
2 years (2002-2004)
Total funding = **£840K** (50% industrial contribution)

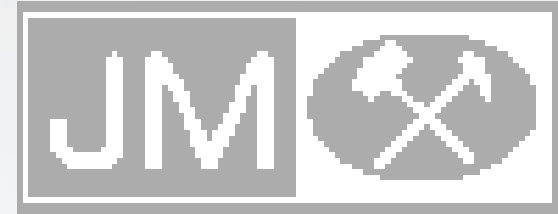
- Facilitate natural gas HCCI using fuel reforming

CHASE (Controlled Homogeneous Auto-ignition Supercharged Engine)
3 years (2004-2007)
Total funding = **£1,539K** (50% industrial contribution)

- Expand gasoline HCCI window



CHAREG/CHASE Project Consortium

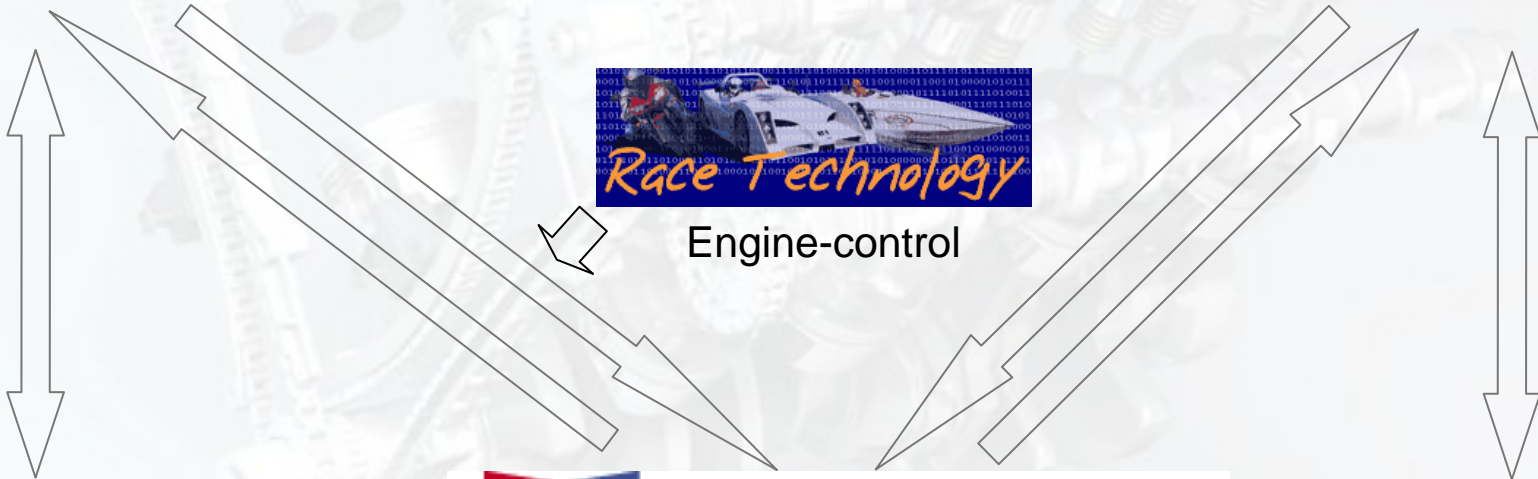


Project leader, engine systems

Reforming catalyst development



Engine-control



Thermal-management



**THE UNIVERSITY
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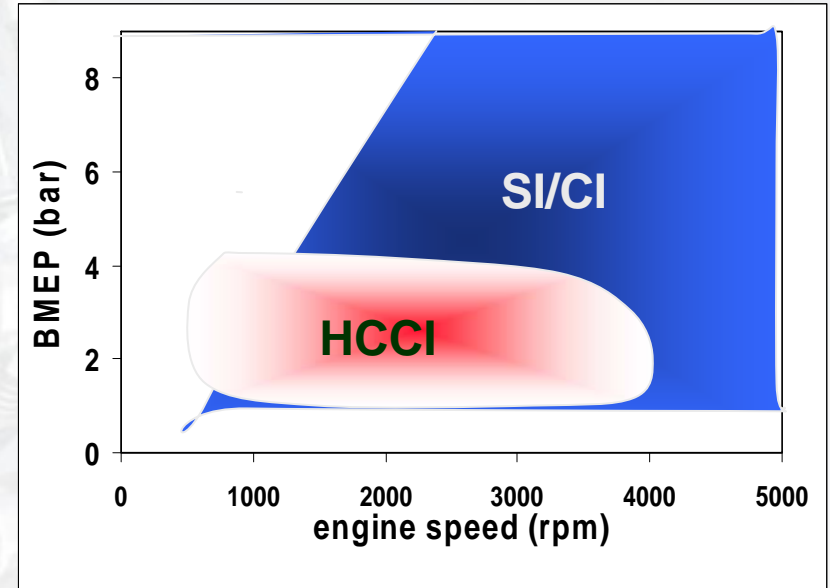
Engine and reforming experiment



Mass-spectrometry

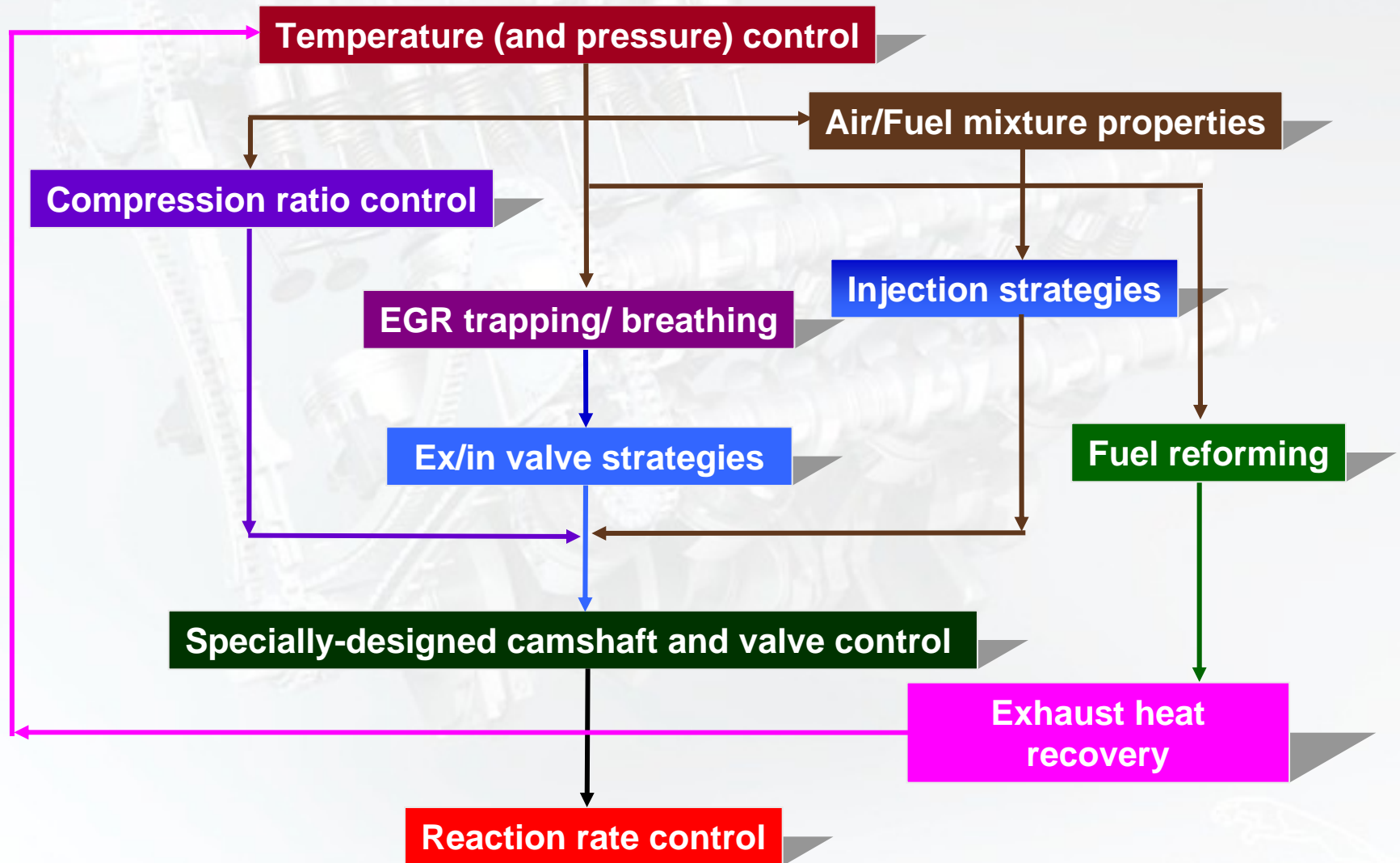
What are the outstanding issues?

- Size of HCCI operation envelopes
- Fuel economy at vehicle level
- Driver transparent transitions between HCCI and SI
- NVH and robustness (Customer interest)
- Cost - engineering (engine upgrade)
material (e.g. special valve-train and control)
manufacturing



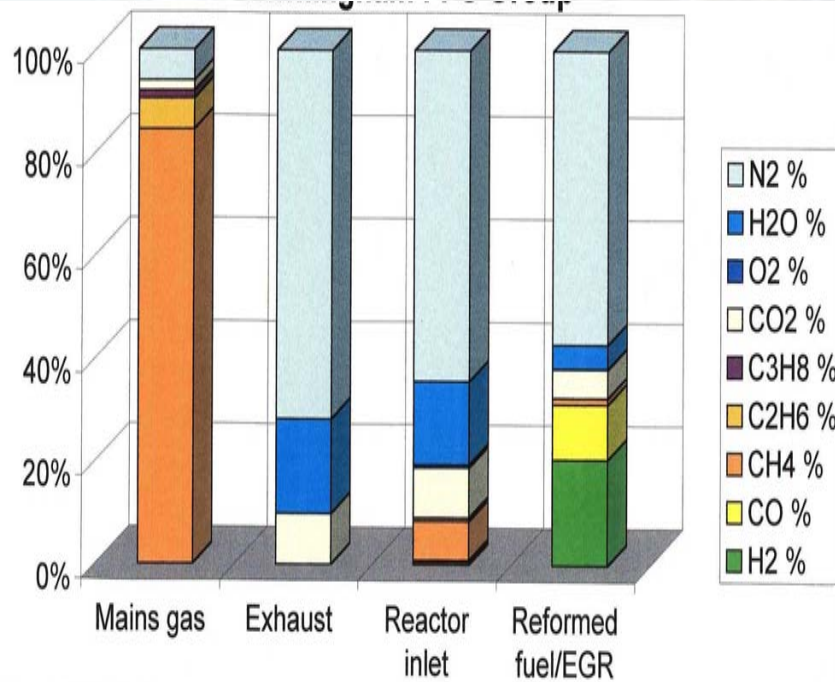
Schematic illustration of HCCI envelope for a vehicle application

Strategies to enable HCCI

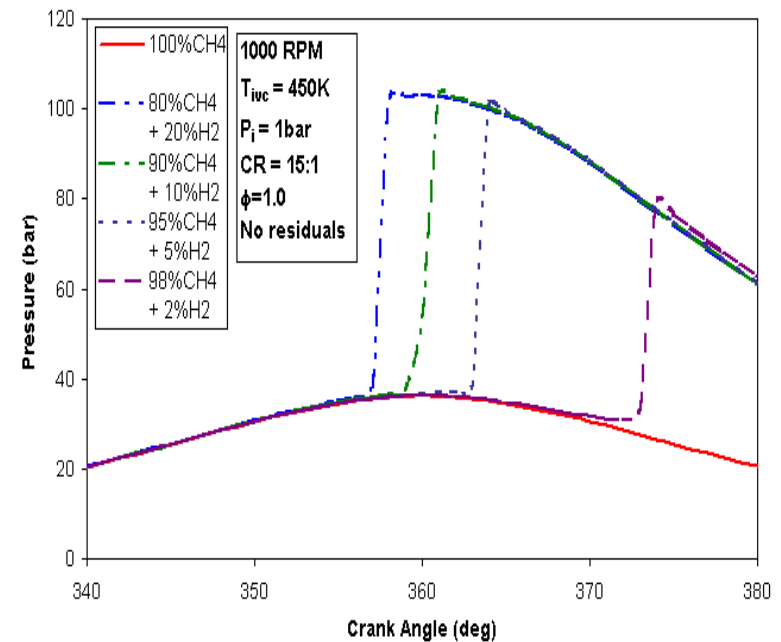


CHARGE Project Concept

Reformed natural gas (test data)

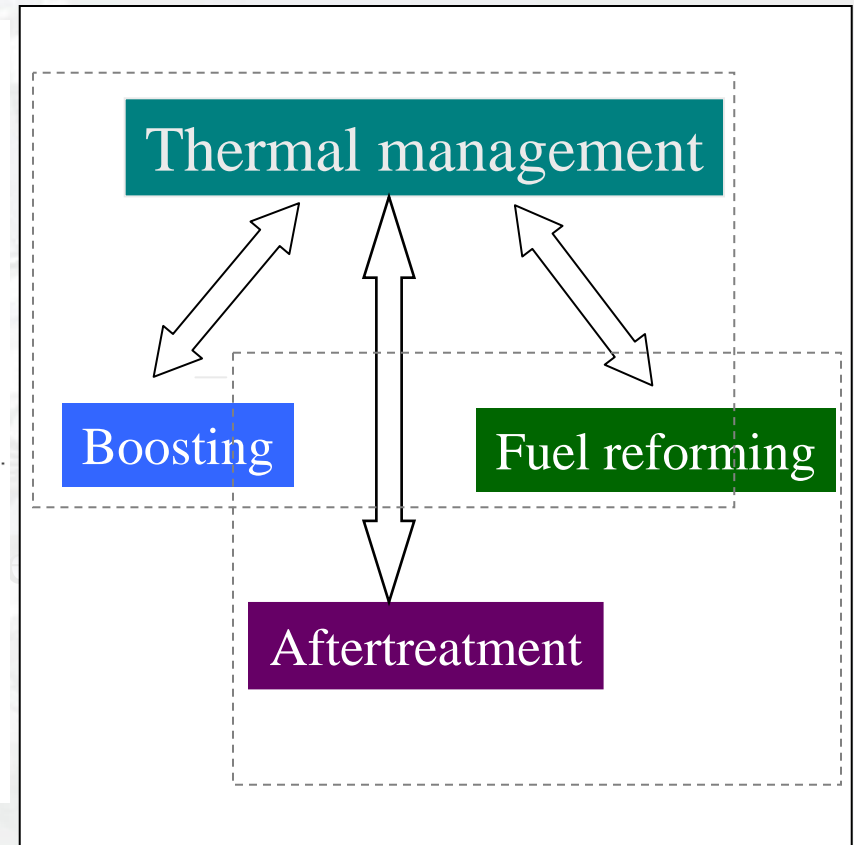
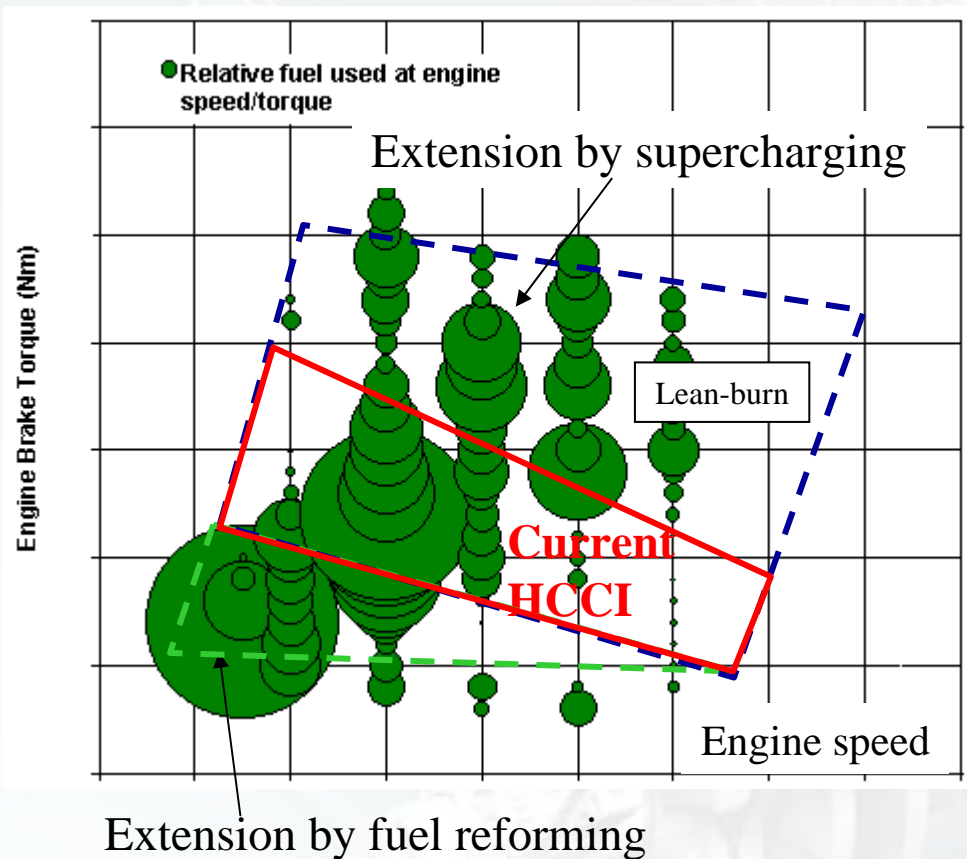


Modelling of the effect of H2 addition



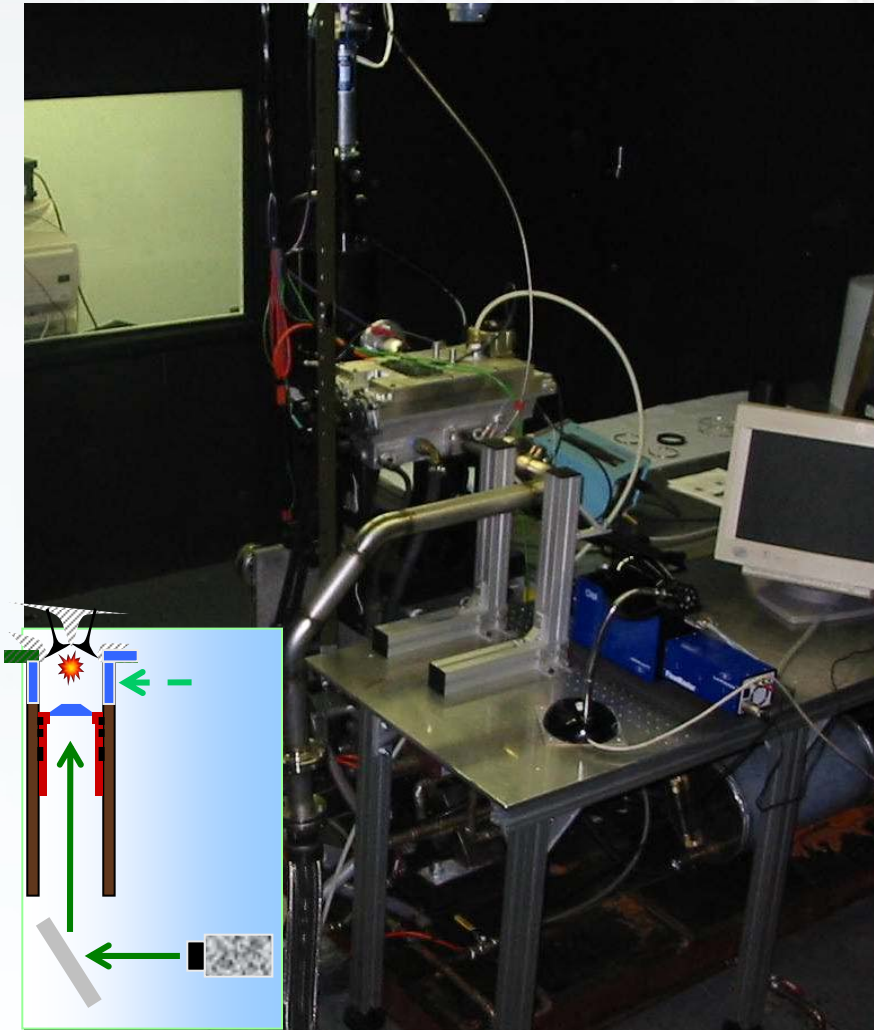
Evaluate the effect of fuel composition and control of engine parameters on the auto-ignition process of natural gas in automotive engines with on-board fuel reforming

CHASE Project Concept



Extend the operating window of HCCI using combination of boosting, exhaust gas fuel reforming, and total thermal management.

CHARGE/CHASE Research Engines (1)

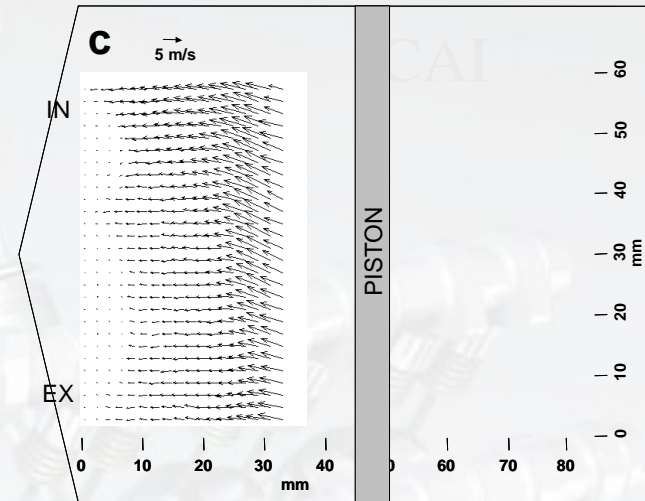
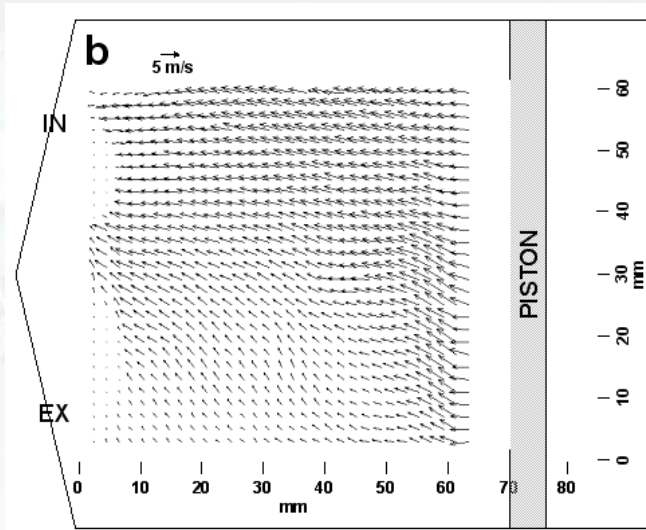


Jaguar Optical engine

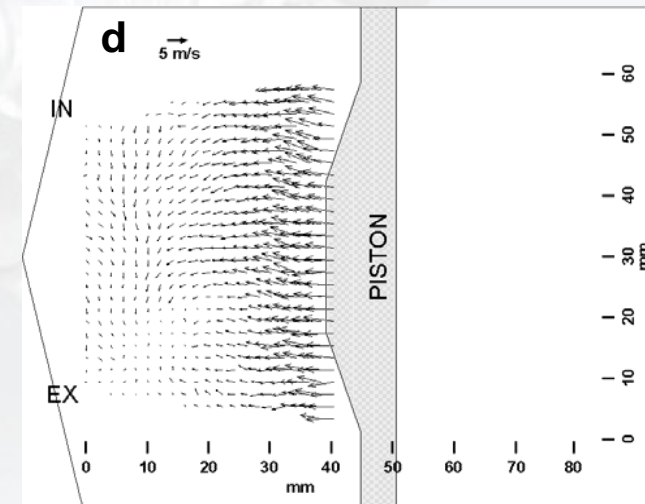
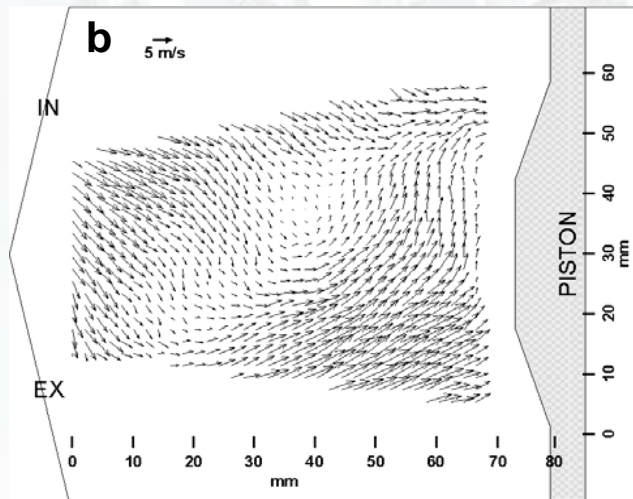
Engine Type	4V, Jaguar V6/V8
Bore	82.5 mm
Stroke	79.5 mm
Compression Ratio	10.5, 11.3 or 14
Fuel system (1)	Liquid – DI or Port injection
Fuel system (2)	Gaseous – Manifold injection
Fuel reformer	Open loop
Intake heating	Electrical, up to 3kw

Flow characterisation with PVO and NVO

Positive
Valve
Overlap



Negative
Valve
Overlap

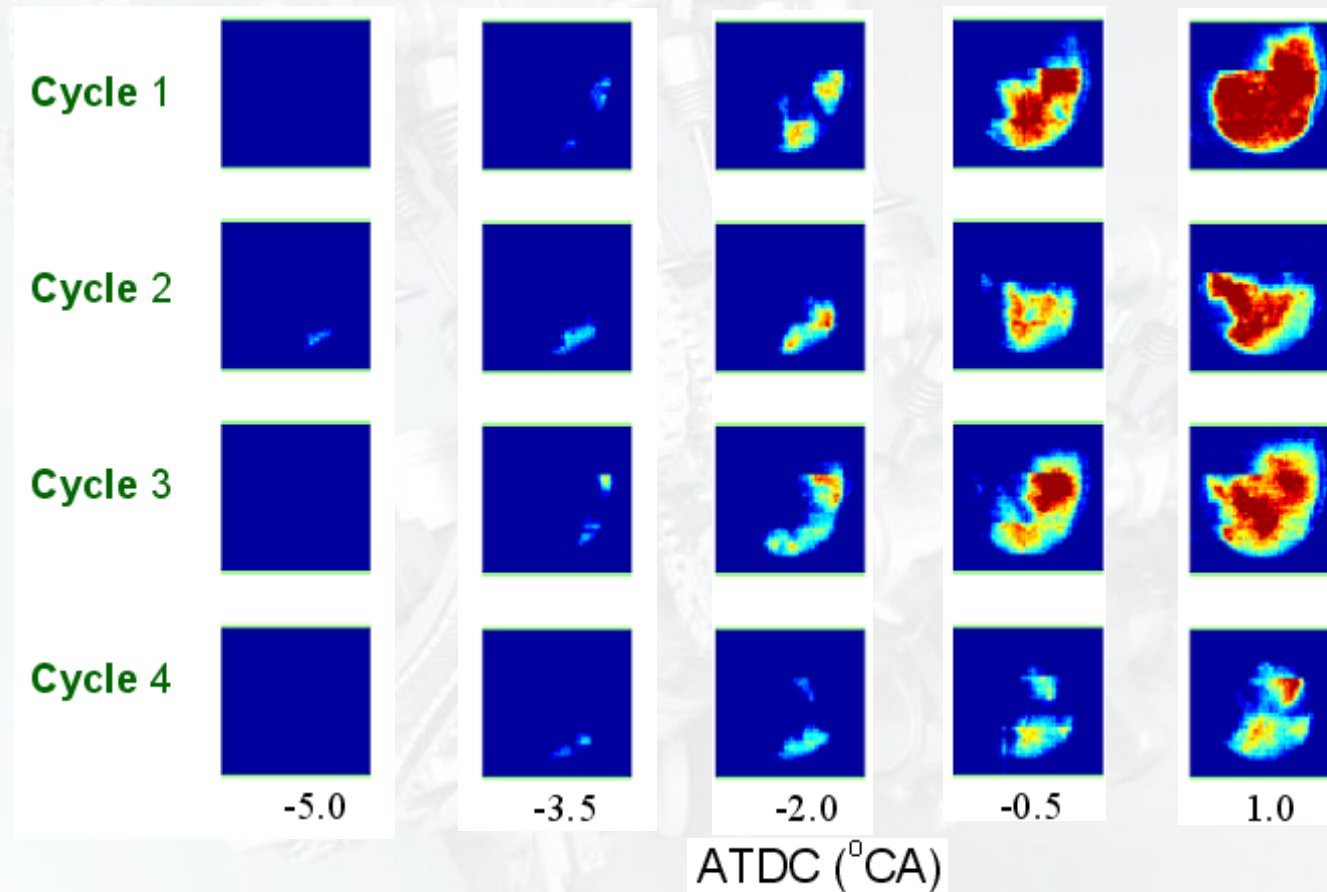


BDC

1000rpm

225 CAD

Flame Propagation?

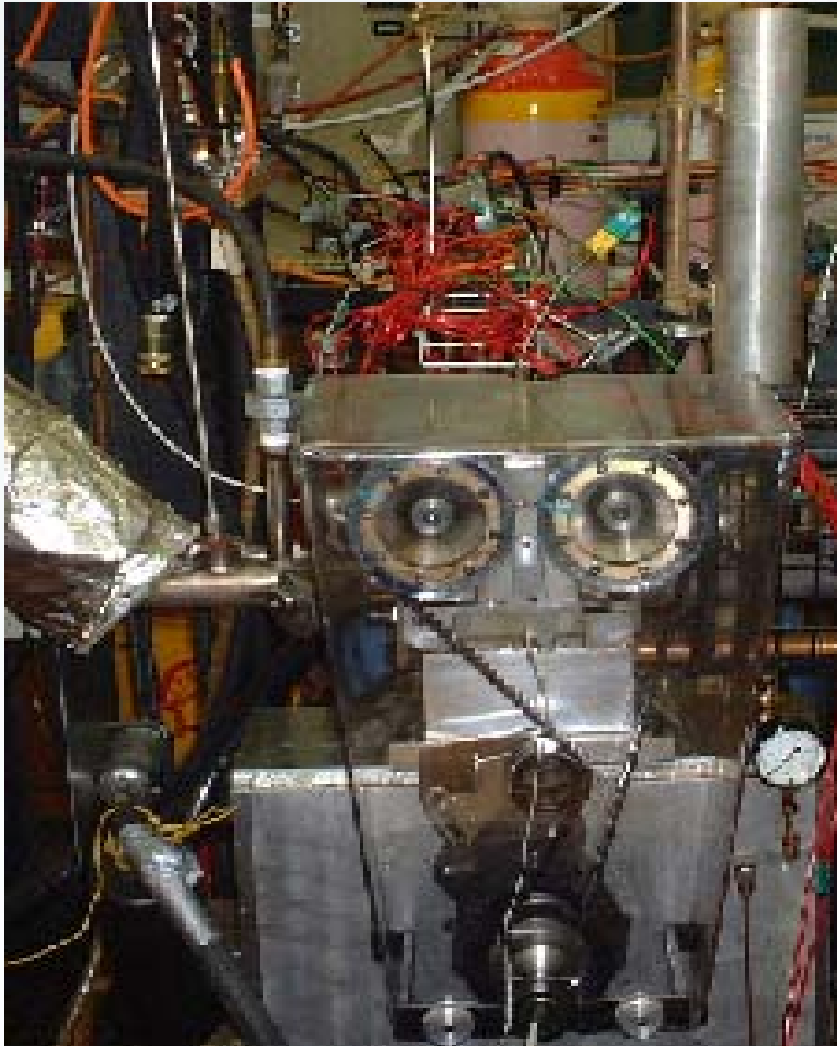


High speed imaging of gasoline HCCI combustion development

⌘ Kodac EM high-speed, 240x50 intensified camera, 4000 fps.

Wilson et al, 2005

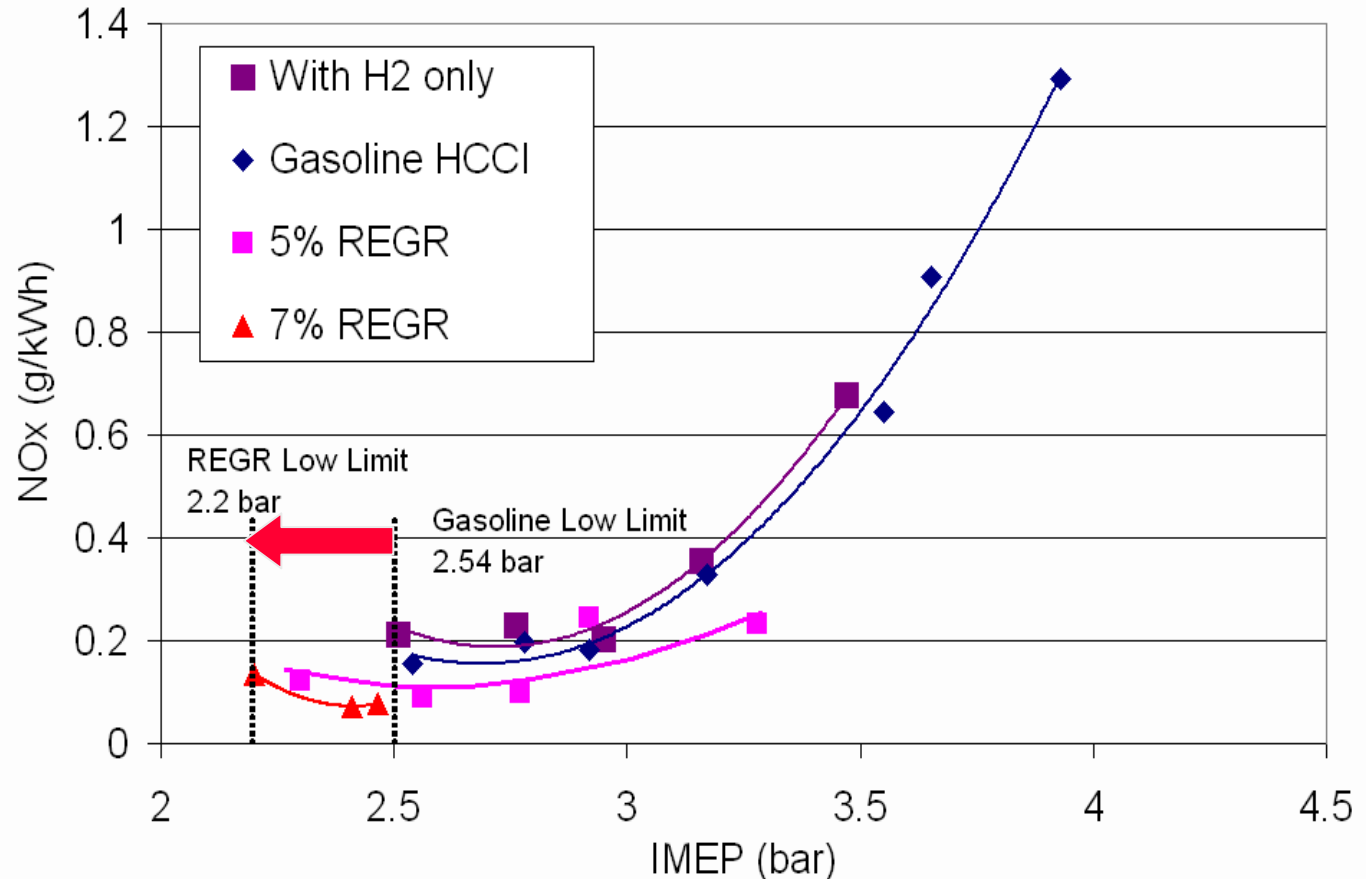
CHARGE/CHASE Engines (2)



Single cylinder Thermal engine

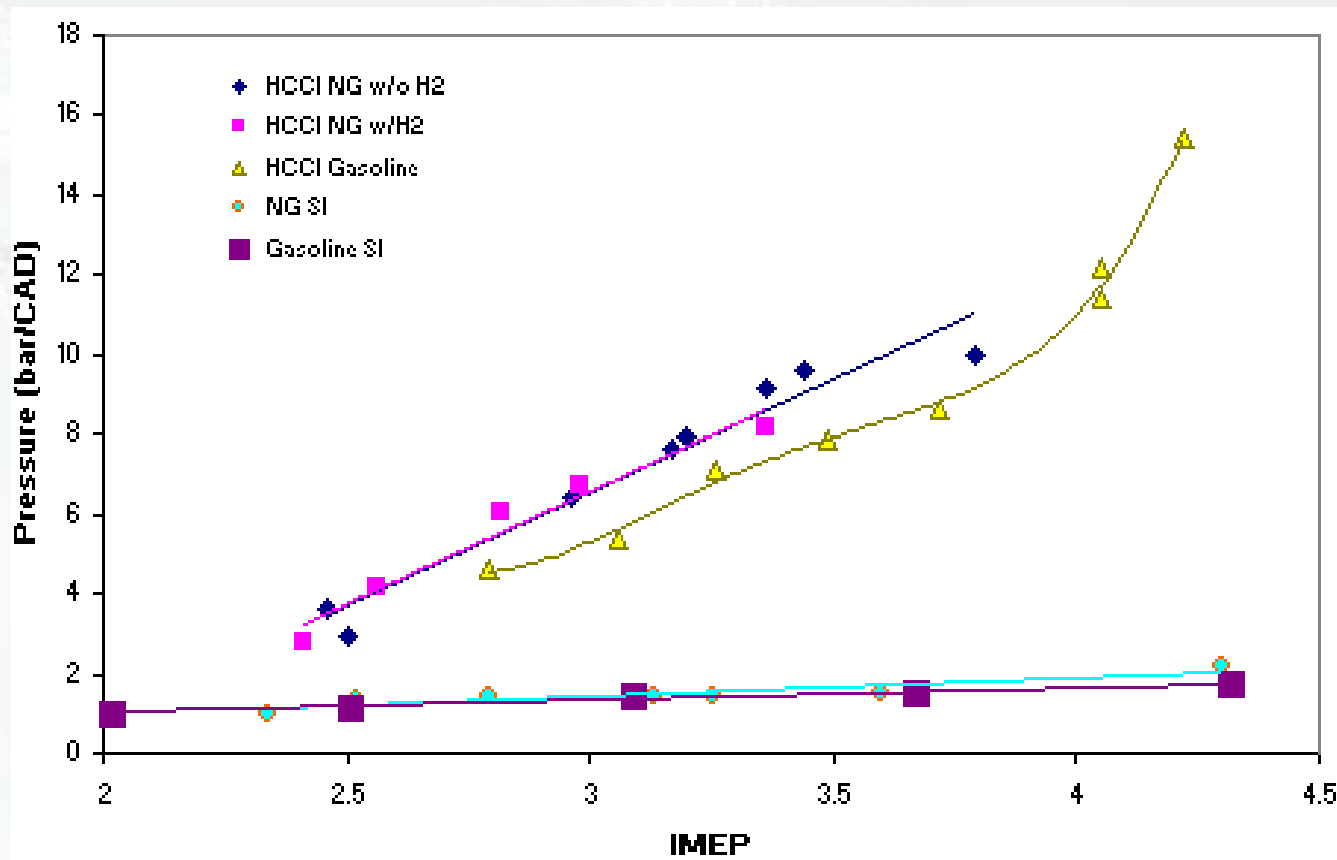
Engine Type	Medusa 4V
Bore	80 mm
Stroke	88.9 mm
Compression Ratio	10.5, 12.5 or 13.3
Fuel system (1)	Liquid – Port injected
Fuel system (2)	Gaseous – Manifold injection
Fuel reformer	Closed loop
Intake heating	Electrical, up to 3kw

Fuel Reforming (1) - NO_x emissions for gasoline HCCI



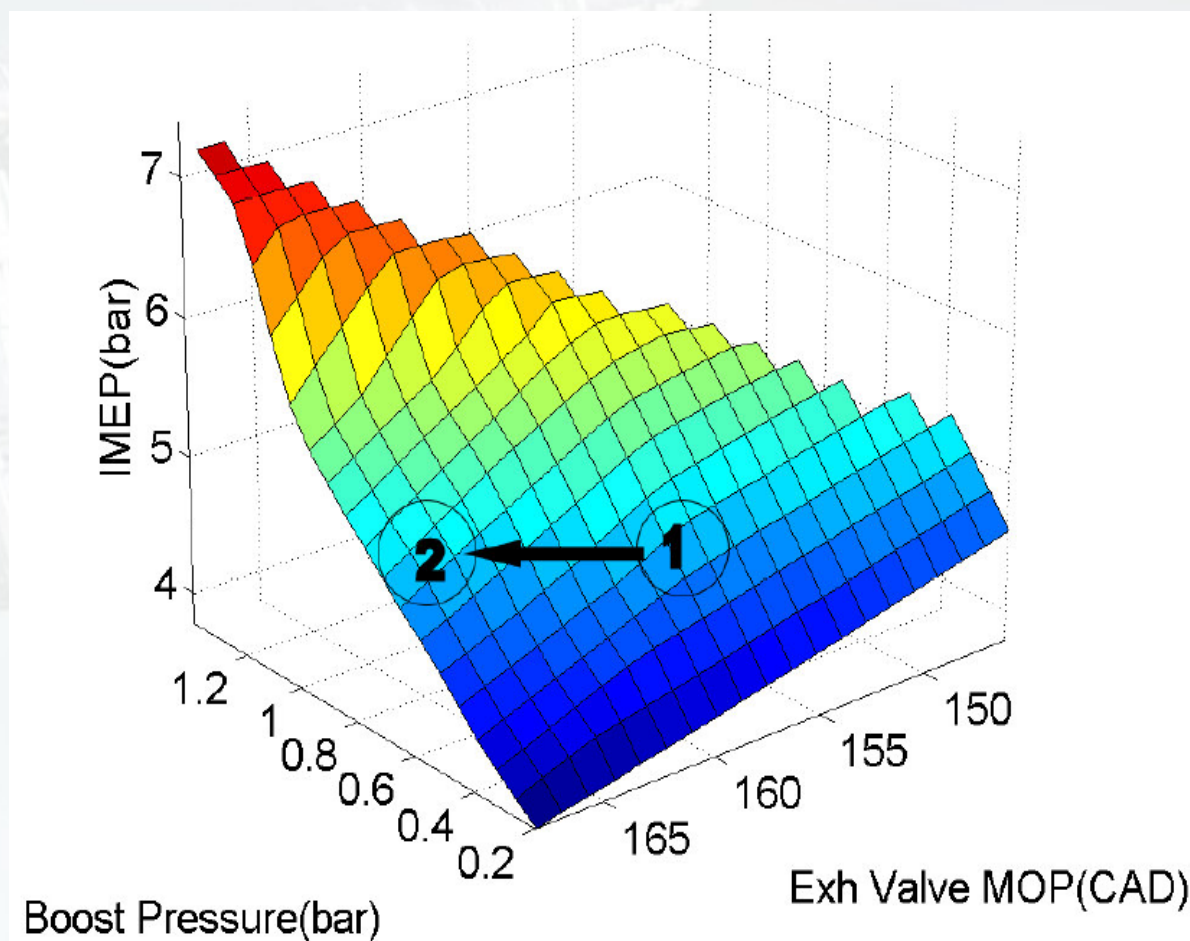
Fuel reforming could extend the low load limit by over 10%

Fuel Reforming (2) – Pressure rise rate with H2 addition



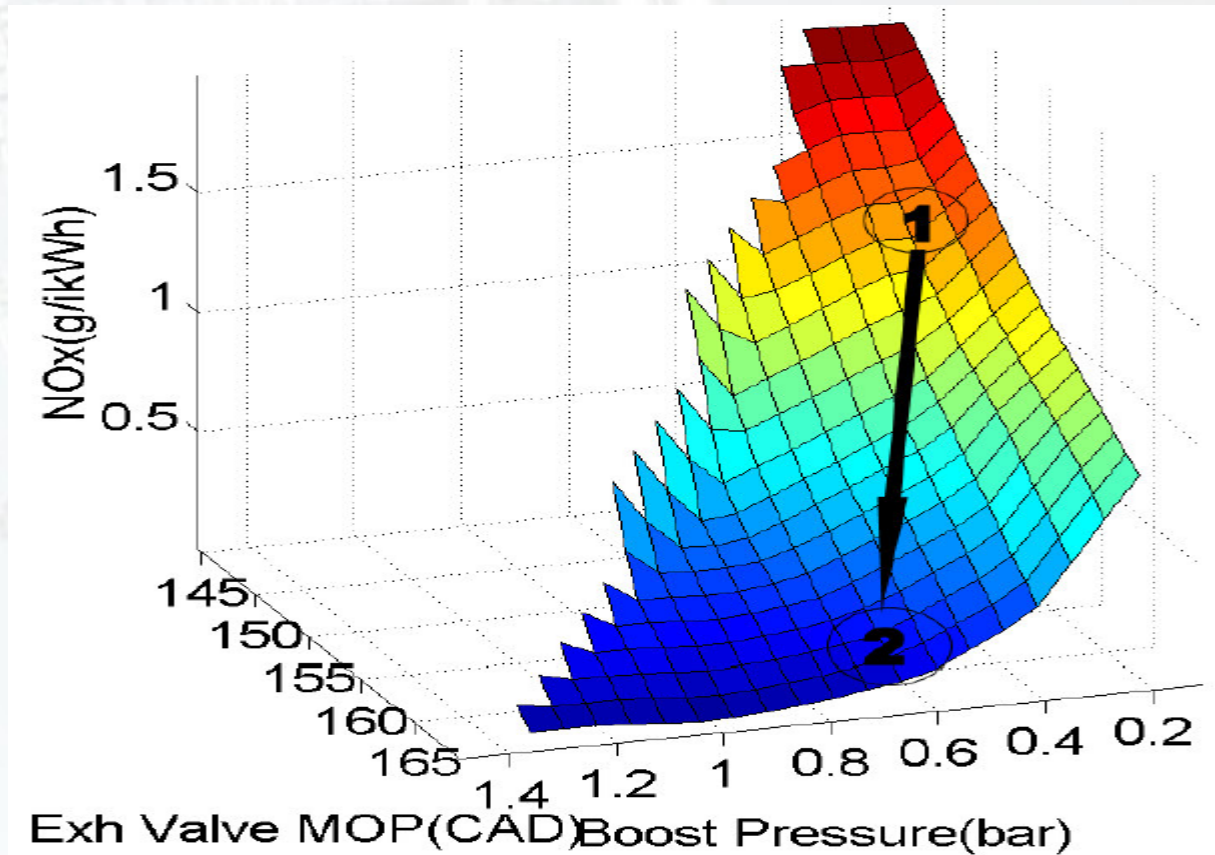
Natural gas HCCI has a higher rate of pressure rise than gasoline HCCI because of the mixing features of the gaseous fuel with EGR and additional effect from intake heating.

Boosting(1) - Valve strategy for minimum NOx emissions



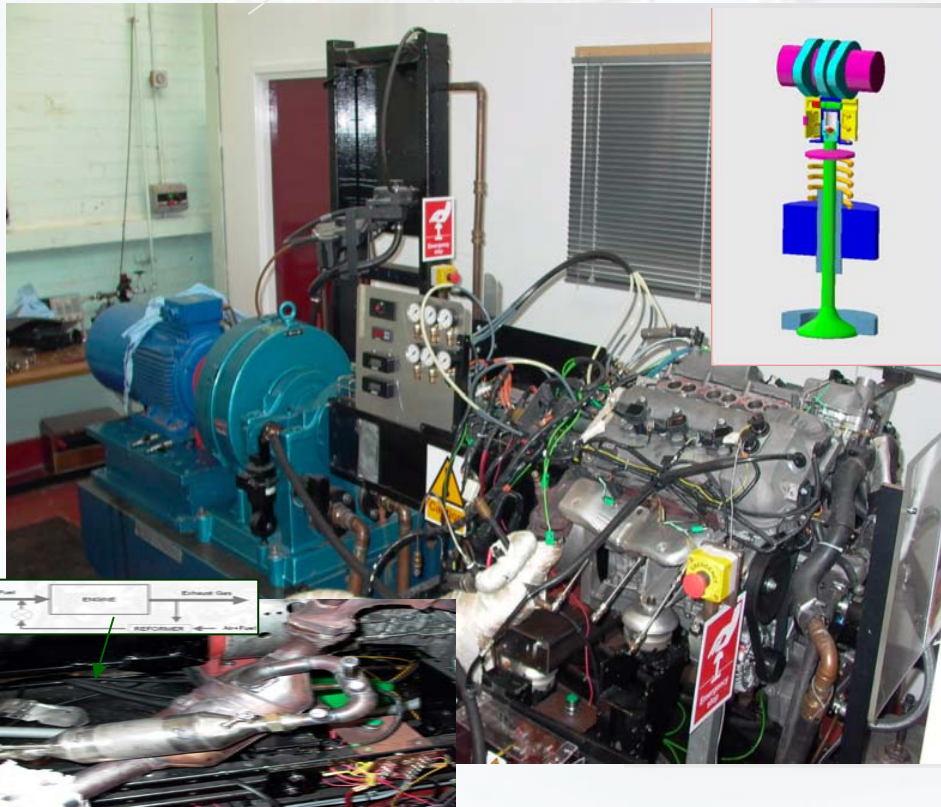
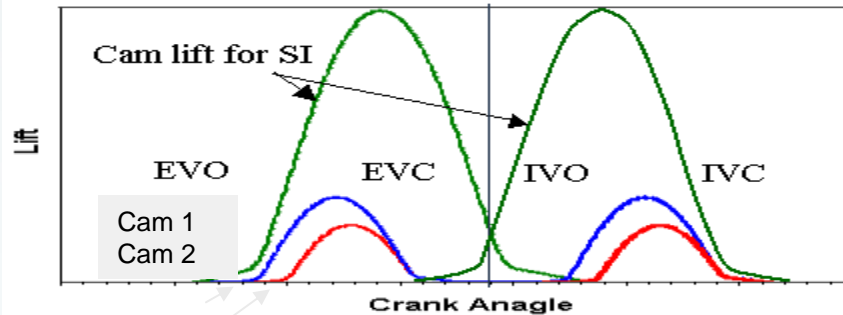
Boosting and EGR selection strategy '1' and '2' have the same engine load but any difference in NOx and fuel economy?

Boosting (2) - NO_x reduction with diluted combustion



Advance exhaust valve closing and increase boosting pressure allow a significant reduction in NO_x emissions with a very little penalty in FC

World 1st dual cam profile switching for HCCI

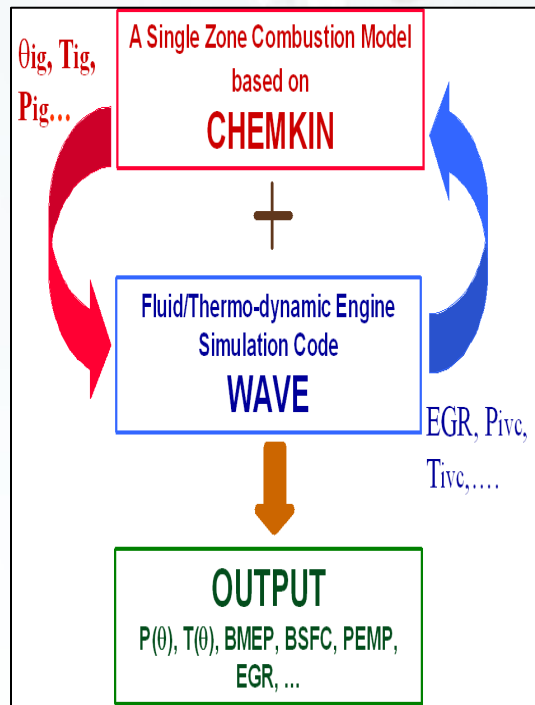


Engine Type	AJV6 – CHARGE (AJV8 - CHASE)
Bore	82.5 (86)mm
Stroke	79.5 (86) mm
Compression Ratio	(9) 11.3 or 13
Fuel system (1)	Liquid – GDI or Port injection
Fuel system (2)	Gaseous – Manifold injection
Fuel reformer	Closed loop
Intake heating	available
Variable Valve Timing	Intake and exhaust

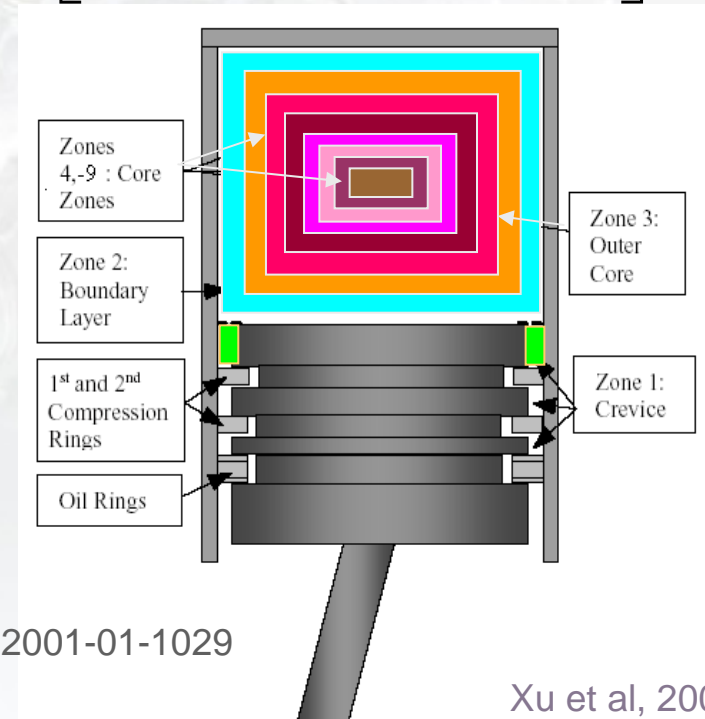
Modelling HCCI engine combustion and operation

Governing equations (similar as in a single-zone model):

$$\bar{c}_{p,i} \frac{dT_i}{dt} = -\frac{1}{\rho_i} \sum_{j=1}^K u_j \dot{\omega}_j M_j - \frac{\dot{Q}_{hli}}{m_i} - \bar{R}_i T_i \left[\frac{1}{V} \frac{dV}{dt} - \frac{\sum_{i=1}^n m_i \bar{R}_i \frac{dT_i}{dt}}{\sum_{i=1}^n m_i \bar{R}_i T_i} \right]$$



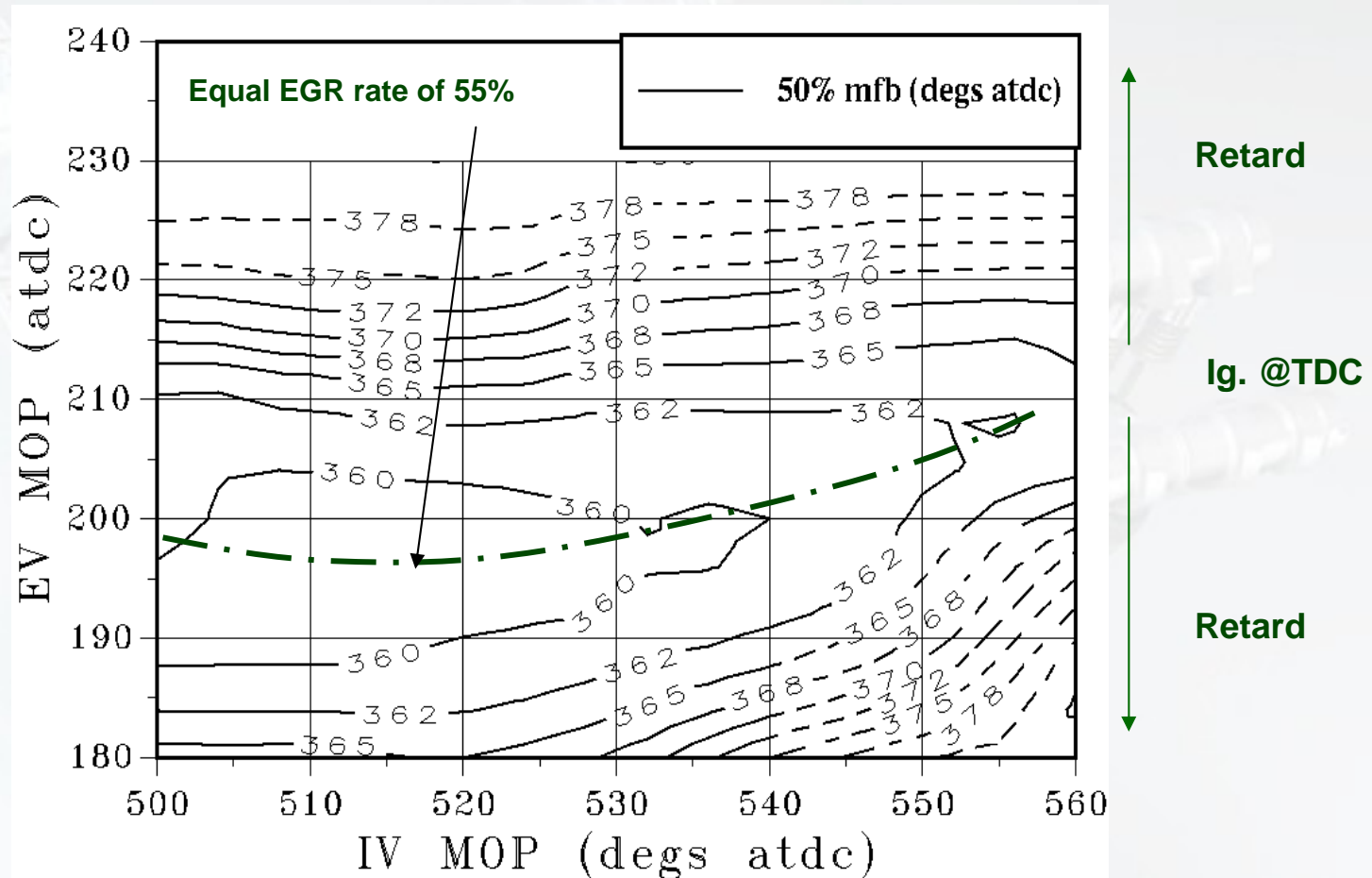
$$P = \frac{\sum_{i=1}^n m_i \bar{R}_i T_i}{V}$$



Ref: SAE 2001-01-1029

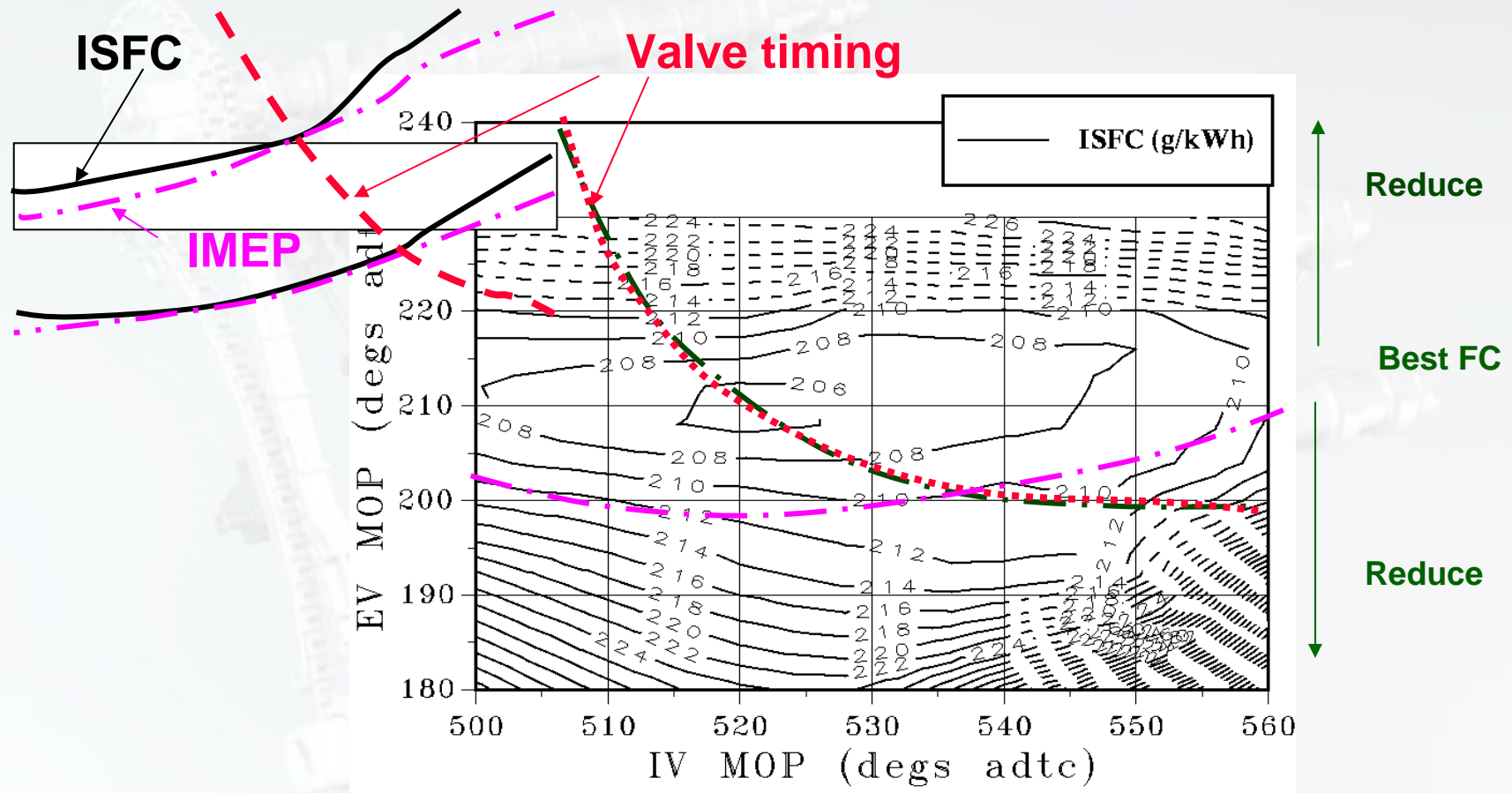
Xu et al, 2005

EGR as combustion phase control



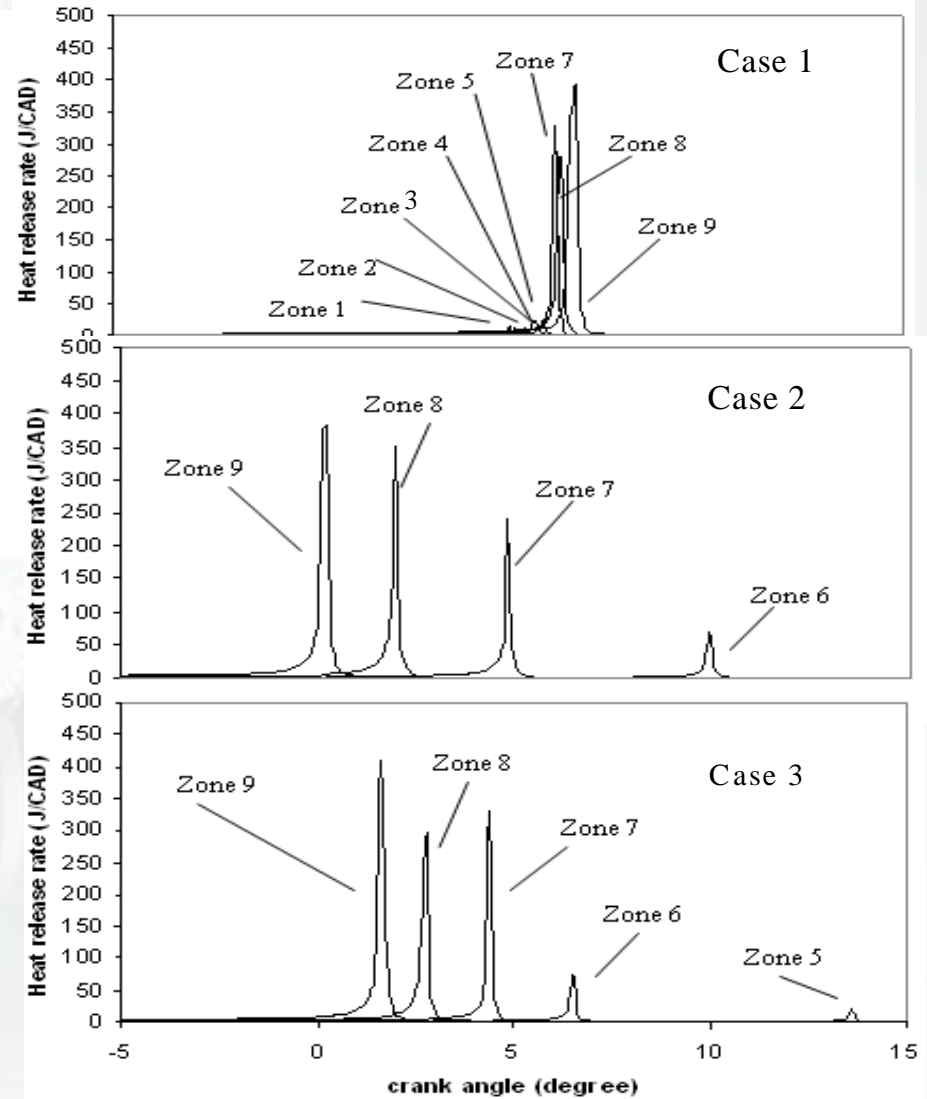
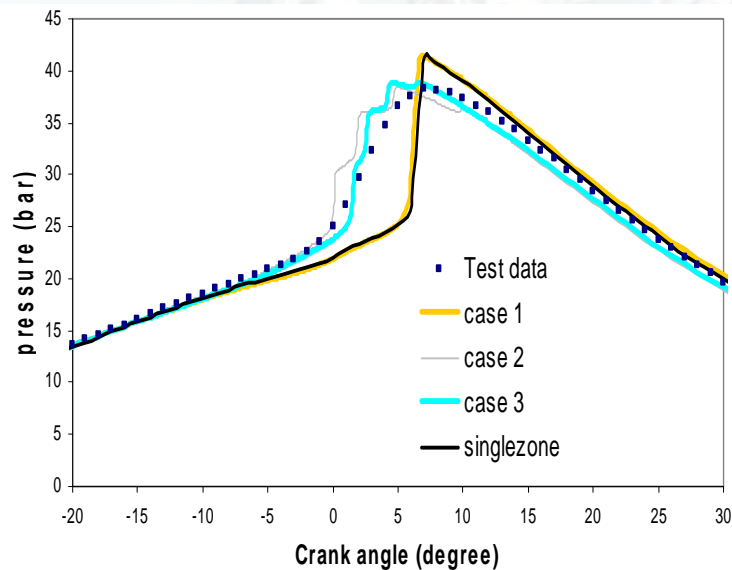
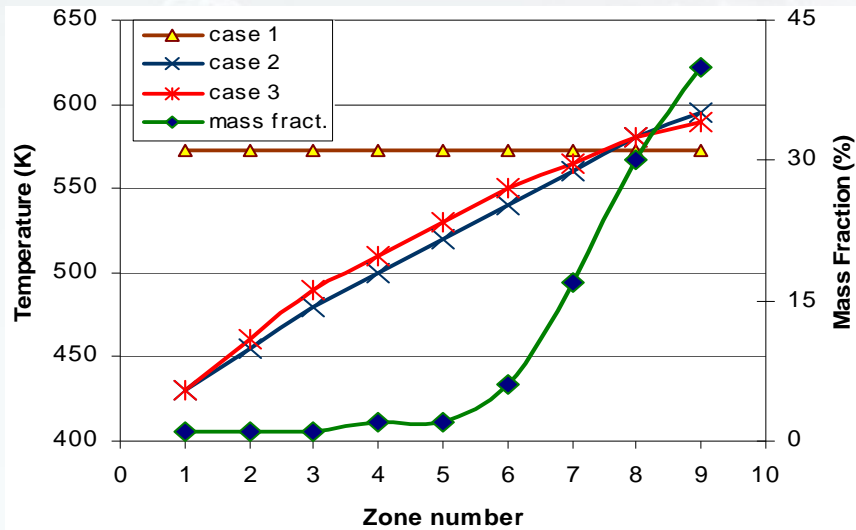
- 55% EGR rate presents the earliest ignition, fastest burning rate and minimum FC line

HCCI Engine modelling

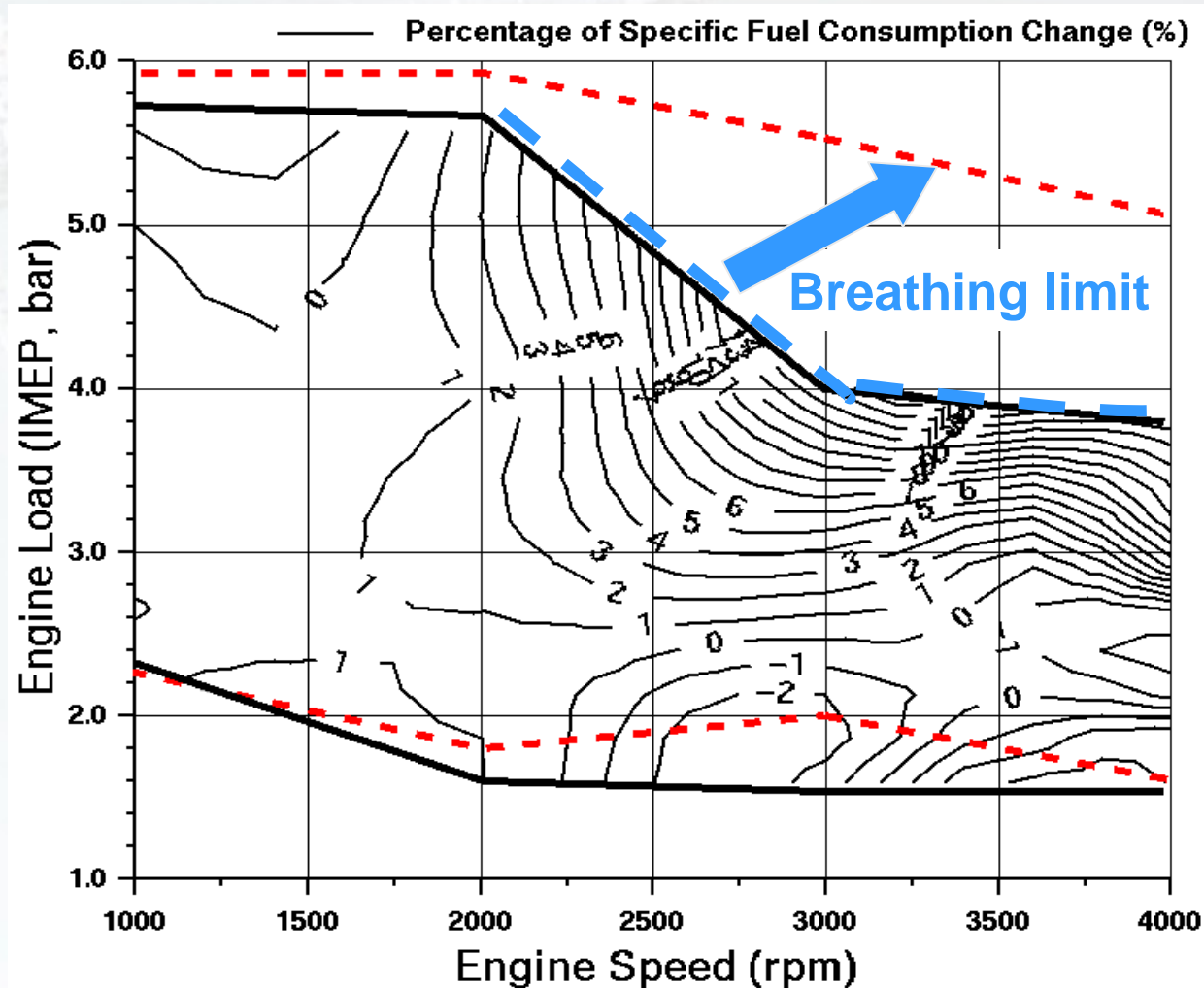


Valve strategy for the best fuel economy with HCCI

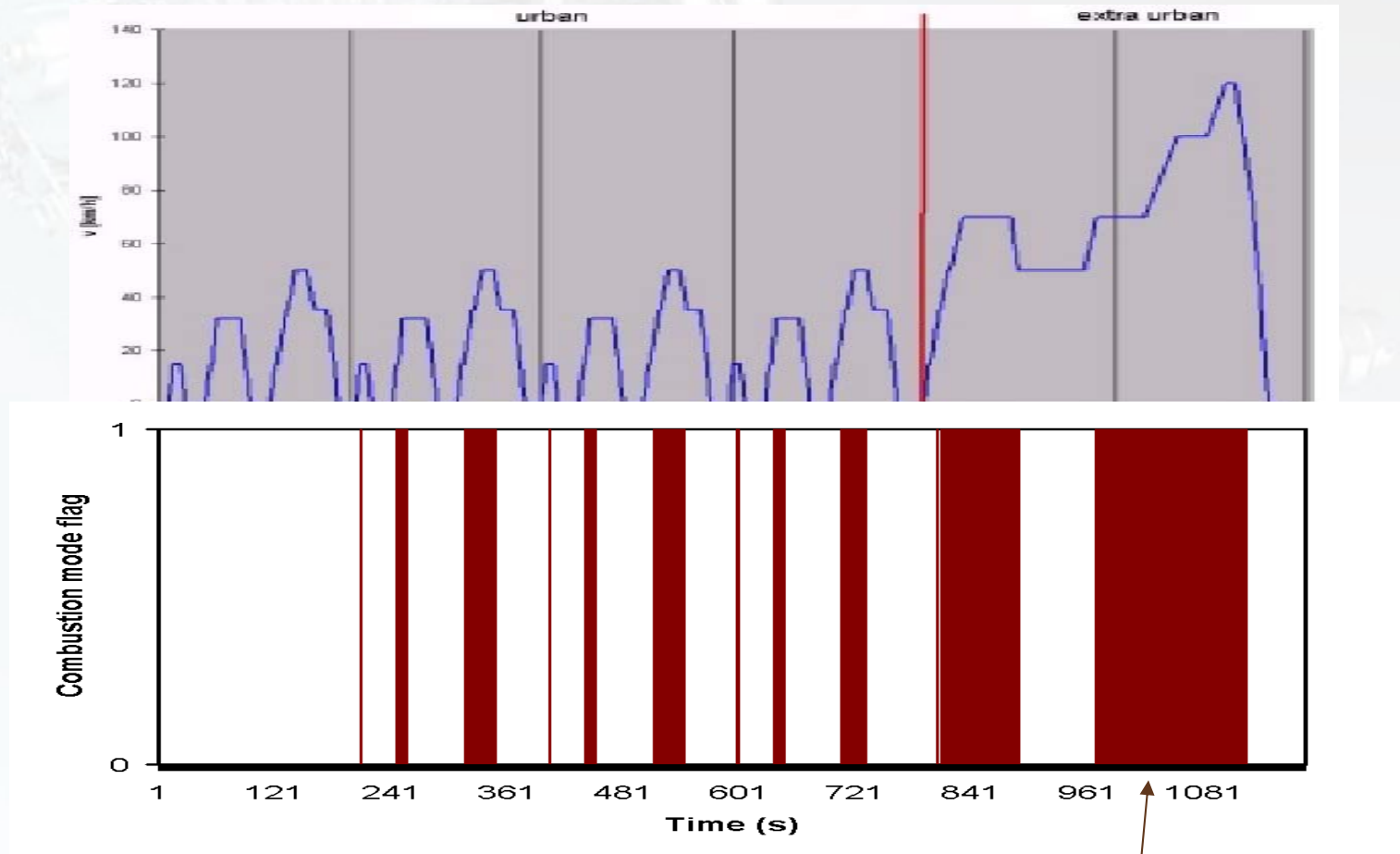
Multi-zone modelling of auto-ignition



Optimisation of valve lift and duration



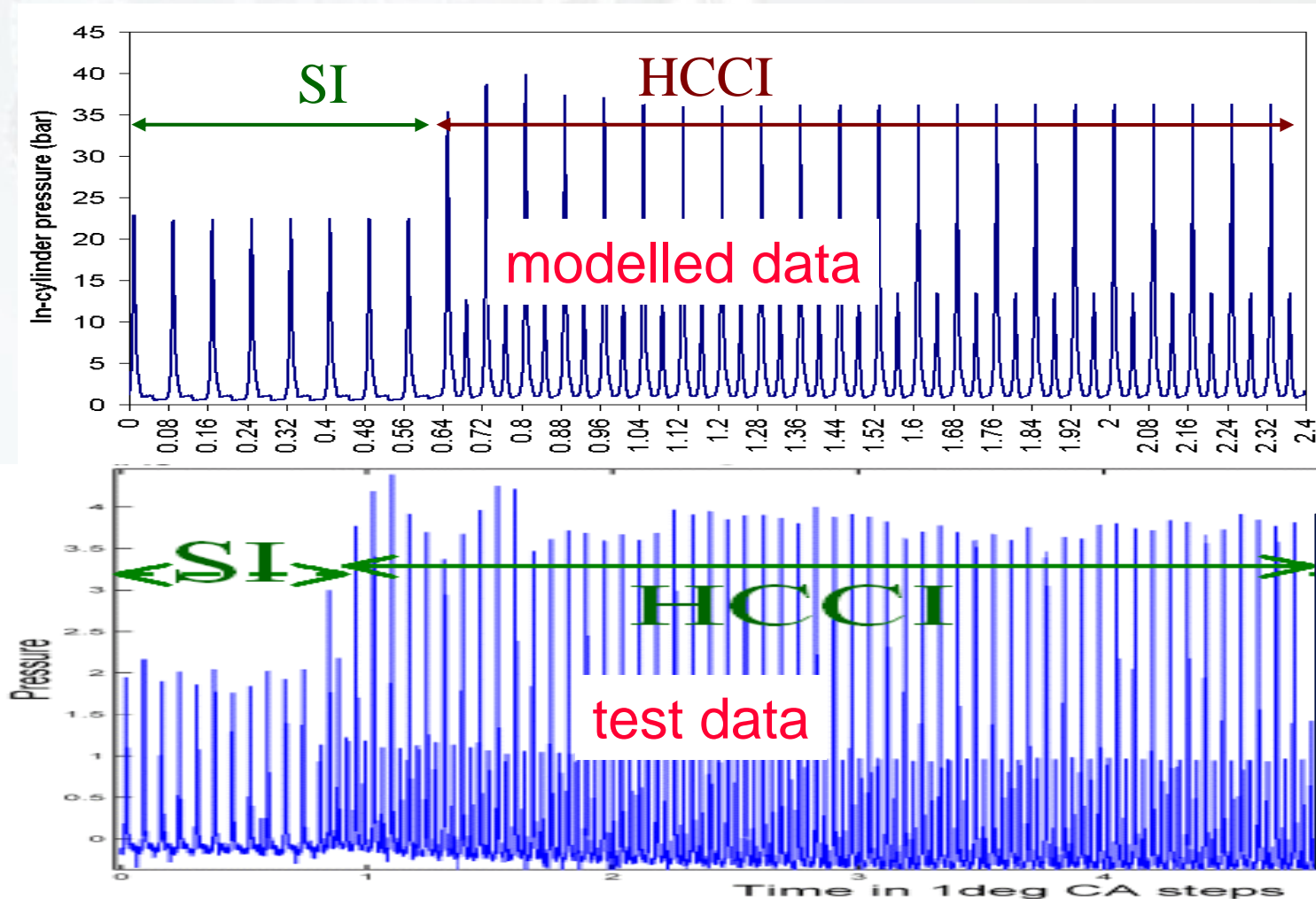
Transitions required for the Jaguar NA V6



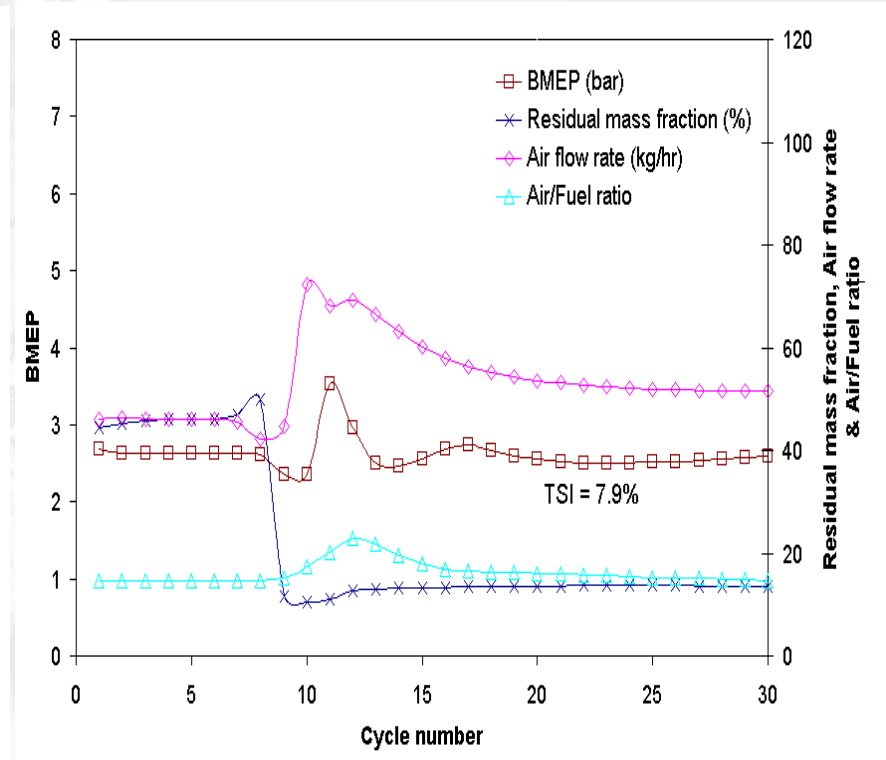
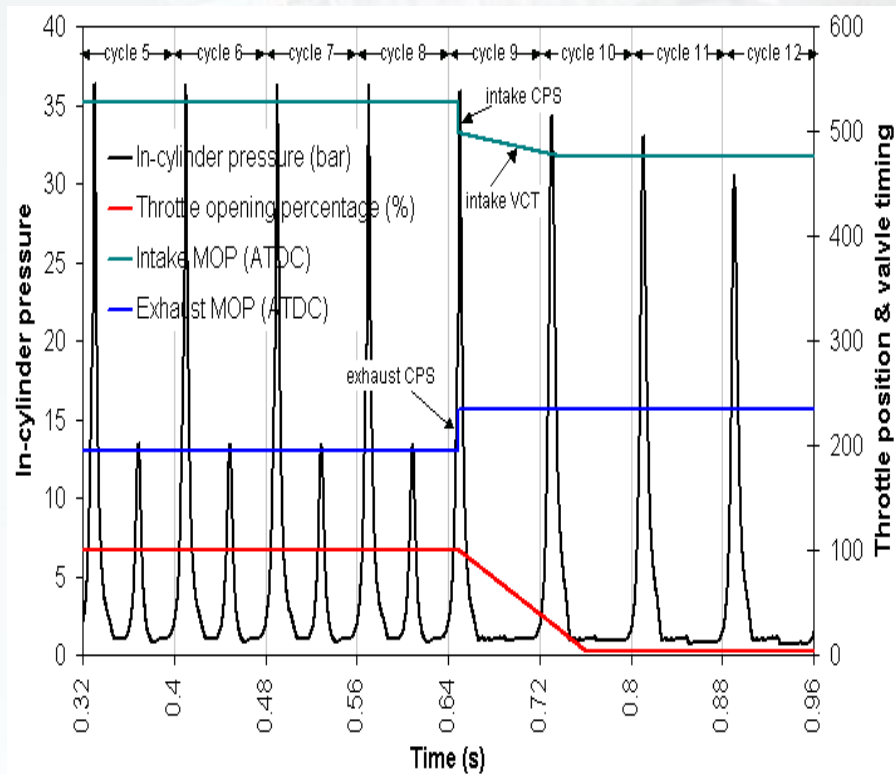
■ CAI/HCCI



Combustion mode transition



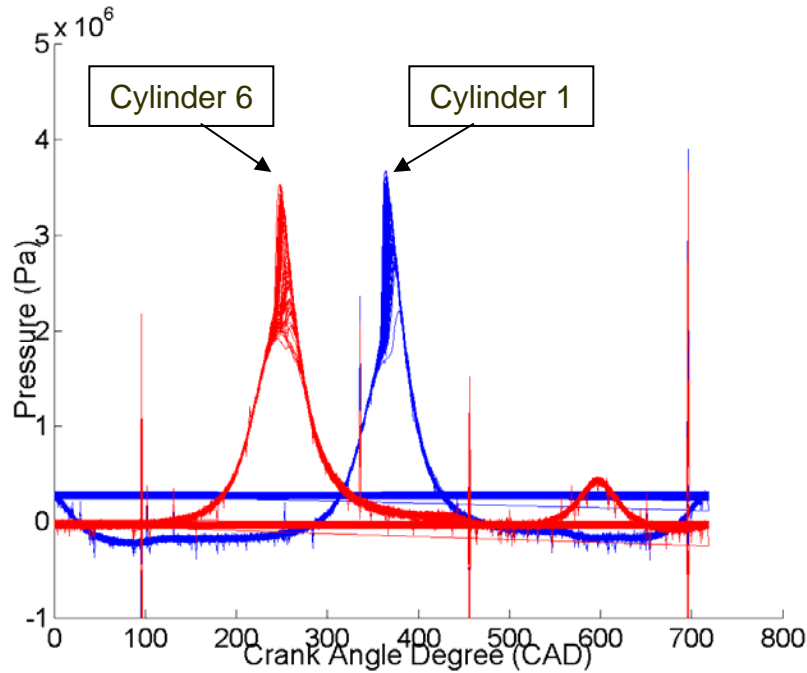
Transition optimisation modelling



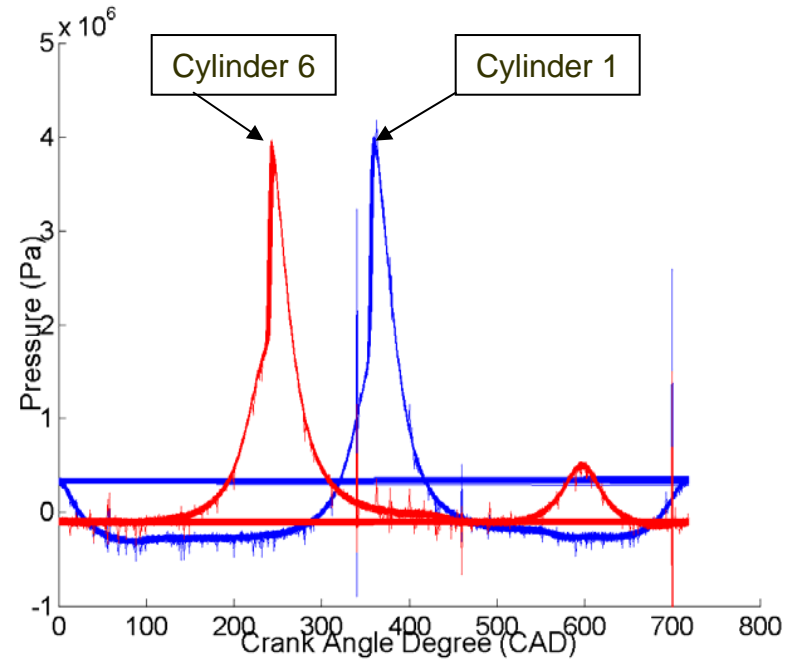
Transient response of throttle, intake and exhaust valves

Variation of BMEP, airflow rate and residual mass fraction

Cycle-by-cycle & cylinder-to-cylinder variations

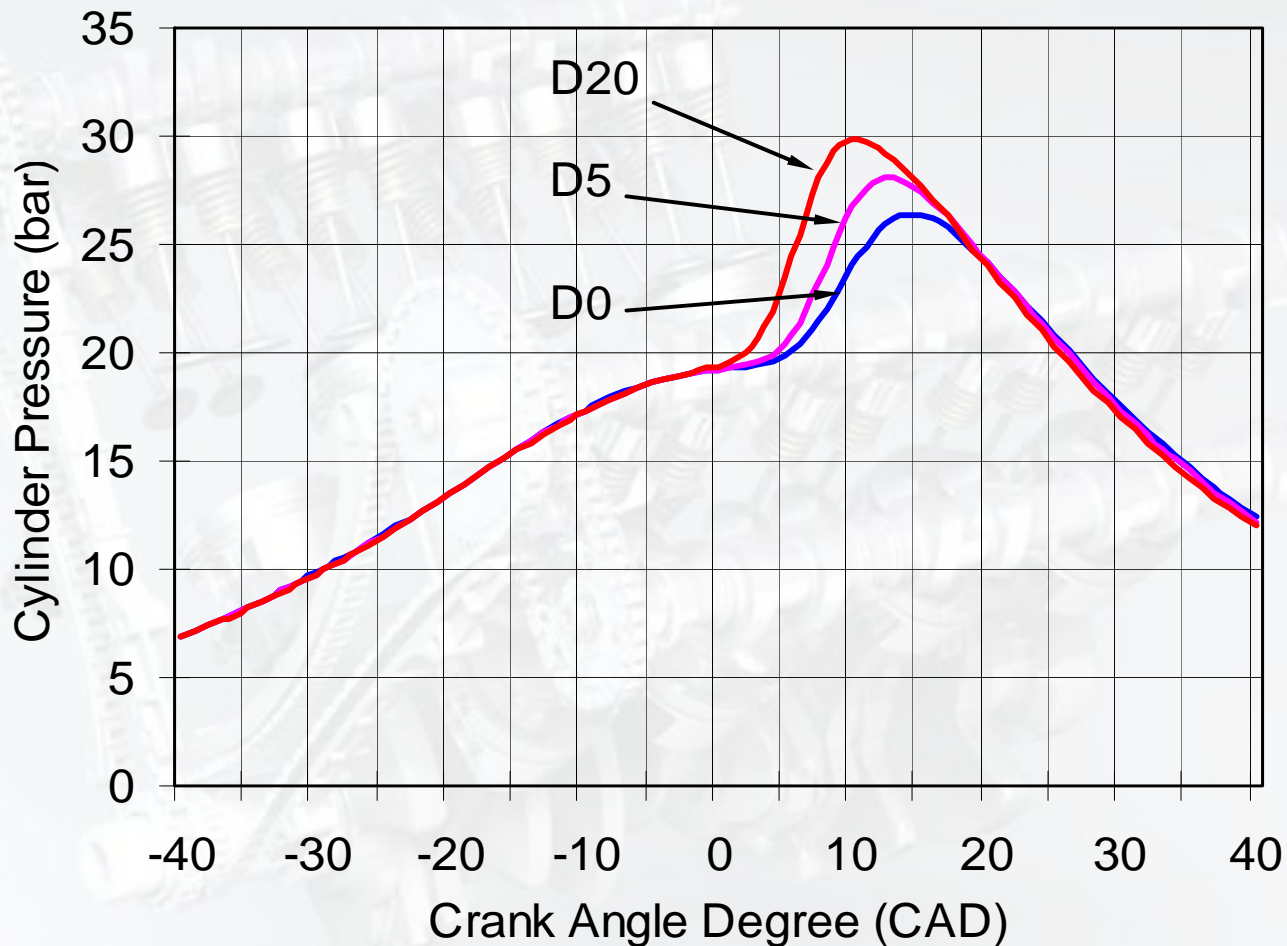


Without reformed gas



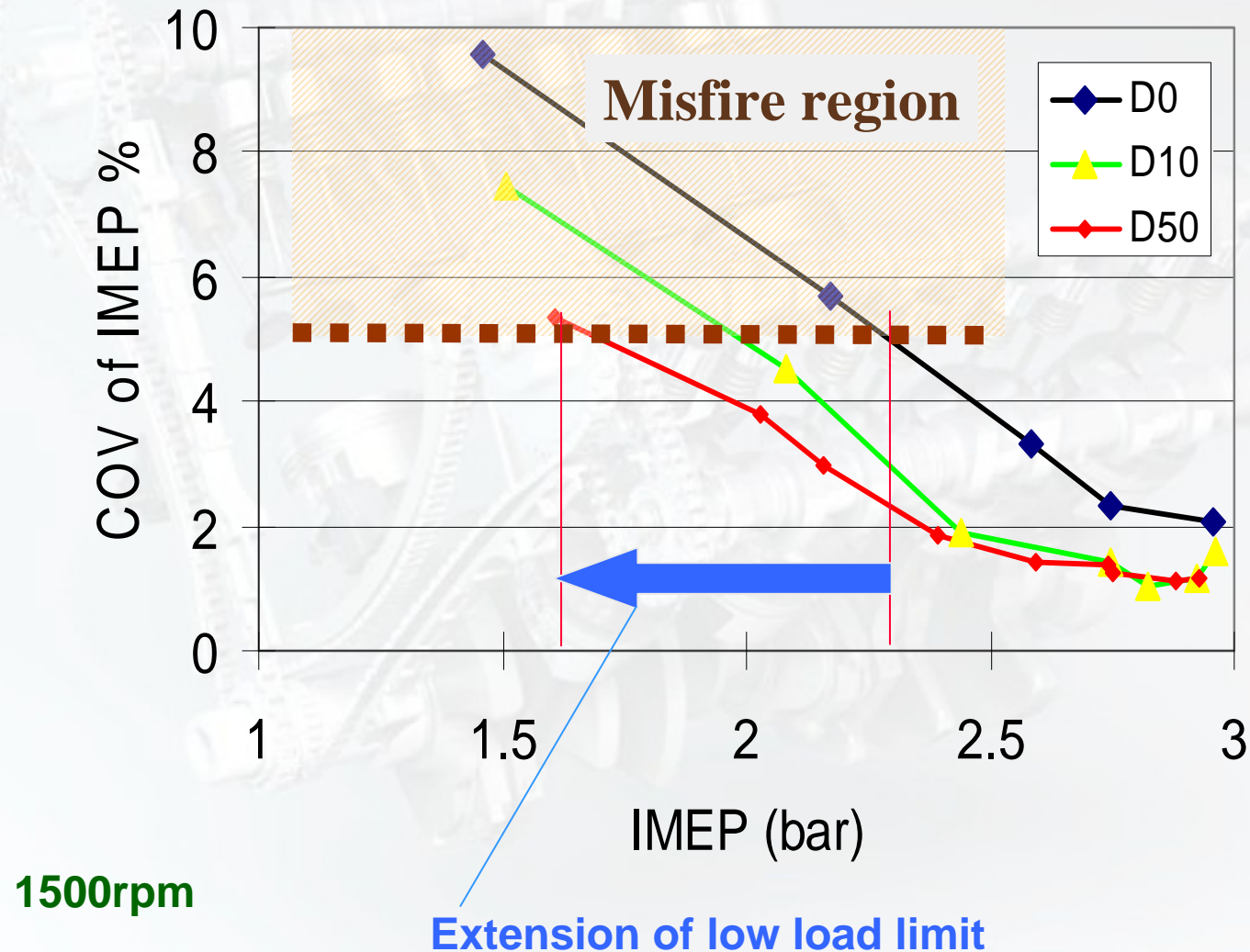
With reformed gas

Gasoline and Diesel blending (1) – Ignition control



1500rpm, $D_x = (100-x)\%$ gasoline + $x\%$ diesel

Gasoline and Diesel blending (2) – expanding HCCL window



HCCI – what are we after?

Conventional Diesel

Comp-ignition

- Fuel with high T_{Ai}
- Late fuelling
- Heterogeneous
- Diffusion flame
- Higher C/R
- **Unthrottled**
- Higher EGR
- **Lean burn**
- L-NO_x
- **Higher FE**
- PM
- ...

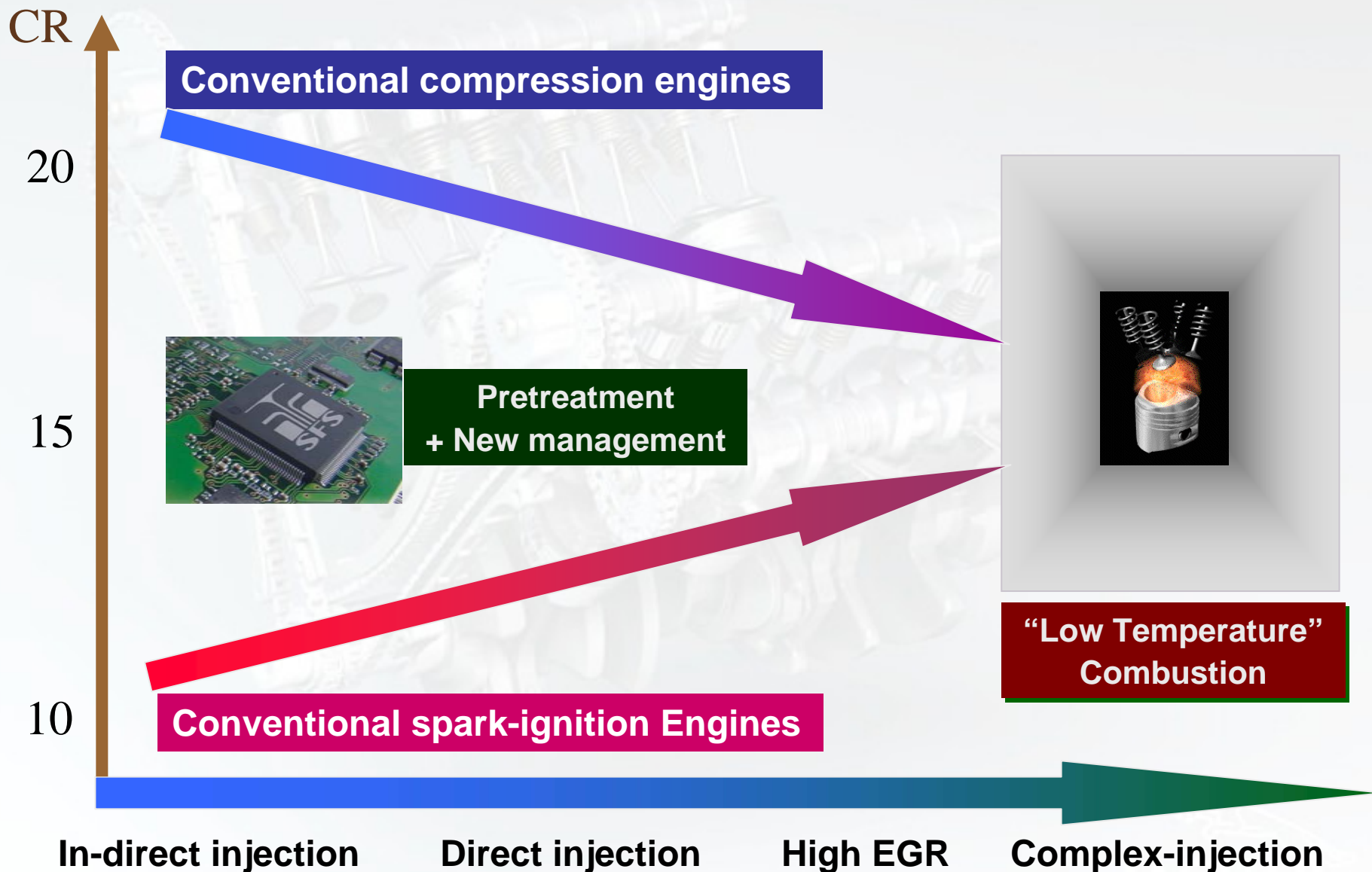
Diesel
Or
Gasoline



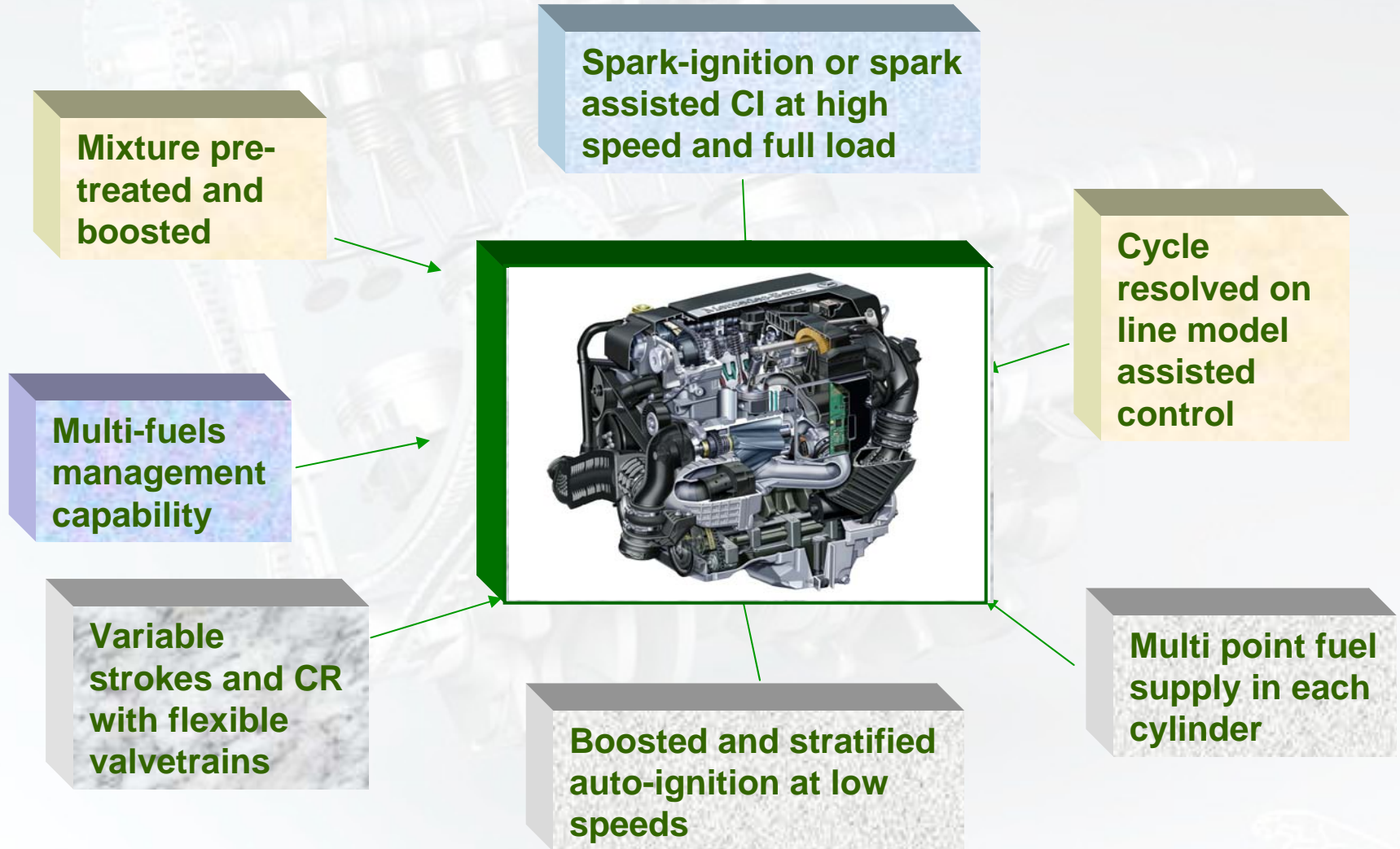
Conventional SI

- Spark-ignition
- Fuel with low T_{Ai}
- Early fuelling
- **Homogeneous**
- **Premixed comb**
- Lower C/R
- Throttled
- Lower EGR
- Stoichiometric
- **No L-NO_x**
- Lower FE
- **Little PM?**
-

Gasoline and Diesel Engine Technologies are emerging



What could the future engine be like?



Conclusions (1)

- The use of HCCI engines with fuels having a high octane number (poor AI quality) employing EGR trapping alone to enable 'controlled auto-ignition' makes it very difficult to achieve cost-effective vehicle level benefits.
- HCCI engines will ideally require air boosting and possibly fuel pre-treatment (in one way or another). Thermal management (system- and in-cylinder resolved) is a very promising technology for expanding the HCCI operating window.
- A new concept of engine control system (e.g. interactive) may be required to handle mixture preparation, ignition and mode transition for the best vehicle performance.

Conclusions (2)

- The author envisages that gasoline and diesel technologies are merging, leading to a new engine system with a combination of “premixed and stratified” combustion.
- Current HCCI developments are to some extent preparing for the advent of other related engine technologies including fully flexible valve-train, variable compression ratio and on-line model based control.
- Future internal combustion engines will burn any combustible liquid and gaseous fuels with a fuel management system. A new type of fuel – ‘dieseline’ is likely to be required to suit the needs of the new combustion system.



Thank you for your attention

Questions?

